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8 Apêndice A

8.1 Equações de Riazi-Daubert para prop. críticas e peso molecular

Temperatura crítica - T_c (K)

$$T_c = 35,9413 \exp[-6.9 \times 10^{-4} T_b - 1,4442 D60 + 4,91 \times 10^{-4} T_b D60]$$

$$T_b^{0,7293} D60^{1,2771} \quad (8-1)$$

Pressão crítica - P_c (bar)

$$P_c = 6,9575 \exp[-1,35 \times 10^{-2} T_b - 0,3129 D60 + 9,174 \times 10^{-3} T_b D60]$$

$$T_b^{0,6791} D60^{-0,6807} \quad (8-2)$$

Volume crítico - V_c (cm³/mol)

$$V_c = 6,2 \times 10^{10} \exp[-7,58 \times 10^{-3} T_b - 28,5524 D60 + 1,172 \times 10^{-2} T_b D60]$$

$$T_b^{1,20493} D60^{17,2074} \quad (8-3)$$

Peso molecular (PM)

$$PM = 42,965 \exp[2,907 \times 10^{-4} T_b - 7,78712 D60 + 2,08476 \times 10^{-3} T_b D60]$$

$$T_b^{1,26007} D60^{4,98308} \quad (8-4)$$

8.2**Cálculo da massa específica (ρ)**

$$\rho = 1000 \frac{PM}{VL} \quad (8-5)$$

$$VL = \frac{R T_c}{P_c} Z_{RA}^{\frac{nr}{nz}} \quad (8-6)$$

$$nr = 1 + \left(1 - \frac{T}{T_c}\right)^{2/7} \quad (8-7)$$

$$nz = 1 + \left(1 - \frac{288,7}{T_c}\right)^{2/7} \quad (8-8)$$

$$Z_{RA} = \frac{PM P_c}{0,999 D60 R T_c} \quad (8-9)$$

$$R = 83,14 \frac{cm^3 \text{ bar}}{mol K}$$

8.3

Fator acêntrico (ω) segundo a definição de Kesler-Lee

$$\omega = \left(-\frac{\ln(P_c)}{1,01325} - 5,92714 + \frac{6,09648}{T_{br}} + 1,28862 \ln(T_{br}) - 0,169347 T_{br}^6 \right) /$$

$$\left(15,2518 - \frac{15,6875}{T_{br}} - 13,4721 \ln(T_{br}) + 0,43577 T_{br}^6 \right)$$

se $T_{br} \leq 0,8$

$$\omega = -7,904 + 0,1352 k_w - 0,007465 k_w^2 + 8,359 T_{br} +$$

$$\frac{1,408 - 0,01063 k_w}{T_{br}} \quad (8-10)$$

se $T_{br} > 0,8$

onde

$T_{br} = \frac{T_b}{T_c}$ é a temperatura de ebulição reduzida

9

Apêndice B

9.1

Simulação numérica

Com o objetivo de verificar a influência da velocidade de rotação do cilindro do viscosímetro na transferência de calor, foi feita uma simulação dos fenômenos envolvidos utilizando o programa CFX. Foi construída uma geometria com as mesmas dimensões do equipamento real, e a partir desta foi gerada uma malha. Foram resolvidas as equações de conservação de massa, quantidade de movimento e energia para o fluido. As hipóteses feitas são as seguintes:

- material do equipamento - aço inoxidável AISI 304
- fluido - óleo de máquina com propriedades descritas no Apêndice A de Incropera e Witt [19]
- interface fluido - ar com coeficiente de película fixo ($h = 50 \text{ W/m}^2\text{K}$)
- sem deslizamento nas paredes e na interface fluido - ar
- condição de contorno na parede do copo - temperatura fixa de 100°C
- temperatura do ar igual a 25°C
- condução de calor no cilindro imerso no fluido

O problema foi resolvido como sendo um caso transiente, onde as condições iniciais são o fluido sem movimento na temperatura de 25°C. As rotações do cilindro consideradas foram: 50, 100 e 150 rpm. Estes são valores típicos do teste com óleo padrão.

A solução é considerada convergida quando os resíduos são menores que 10^{-6} .

Os resultados obtidos são qualitativos somente, e servem somente para indicar uma tendência, uma vez que não foi feito um teste de malhas, e o coeficiente de película se manteve fixo.

As figuras (9.1) e (9.2) mostra o perfil de temperaturas no caso com rotação igual a 100 rpm.

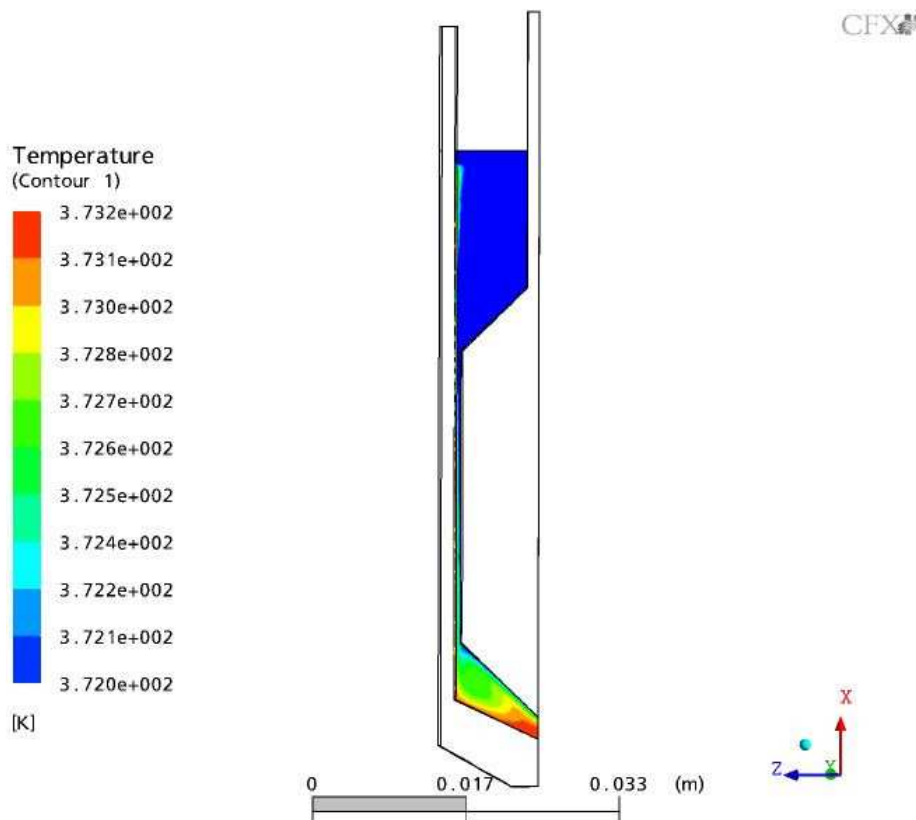


Figura 9.1: Perfil de temperaturas do fluido a 100 rpm

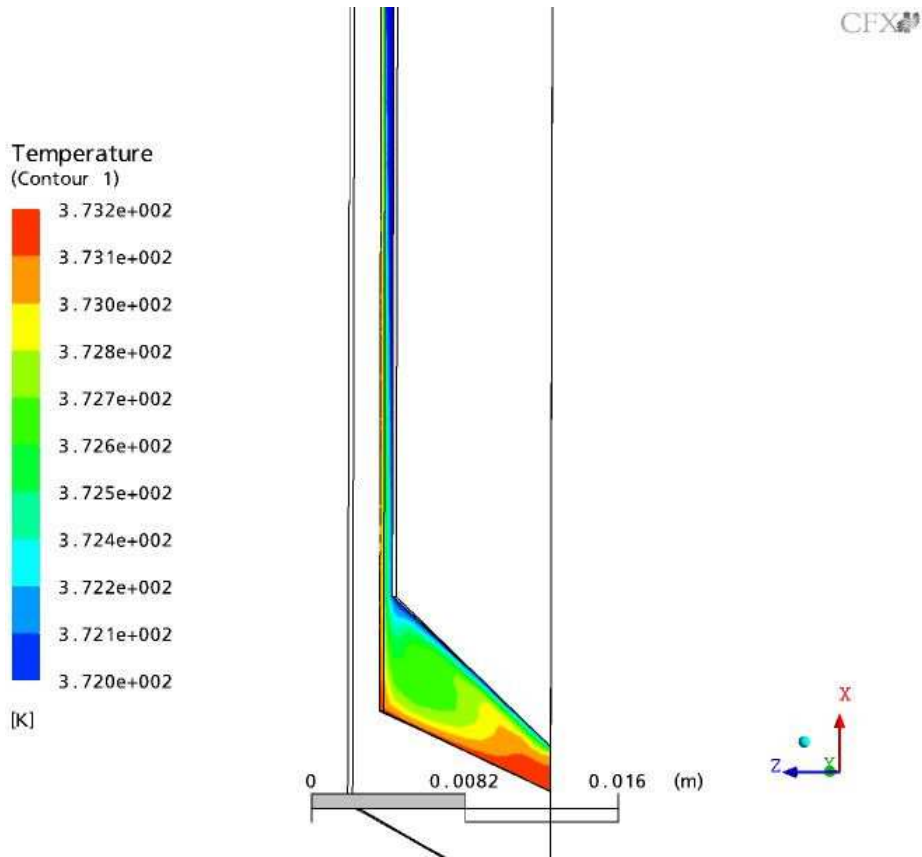


Figura 9.2: Perfil detalhado de temperaturas do fluido a 100 rpm

As figuras (9.3) e (9.4) mostram o perfil de temperaturas no caso com rotação igual a 150 rpm.

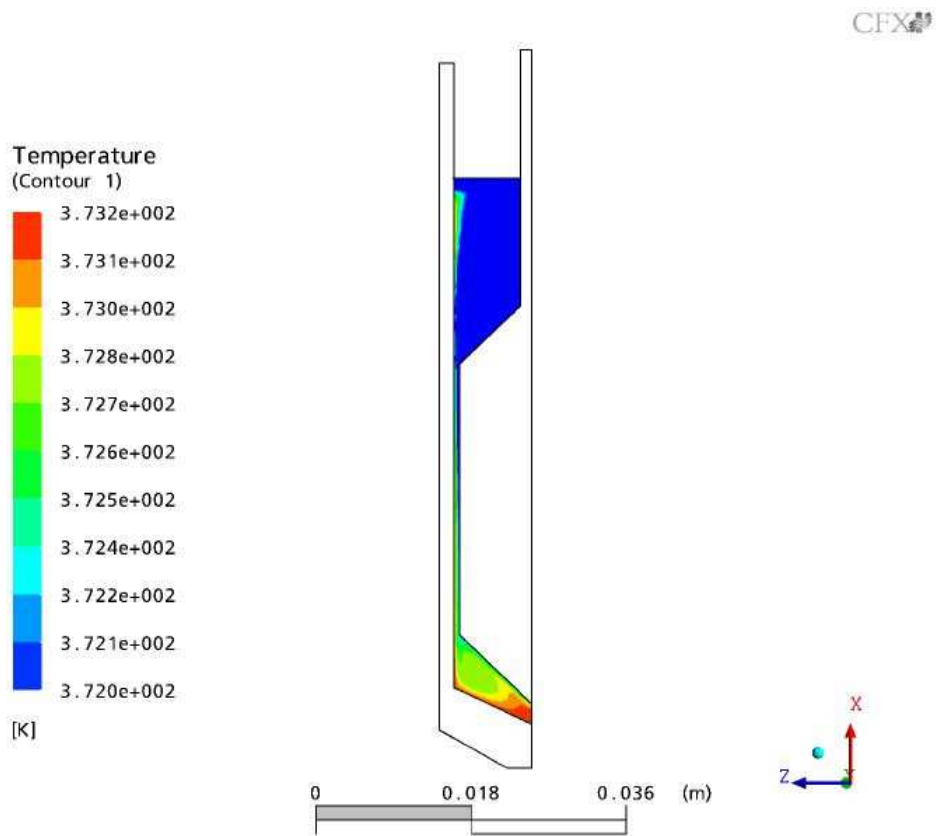


Figura 9.3: Perfil de temperaturas do fluido a 150 rpm

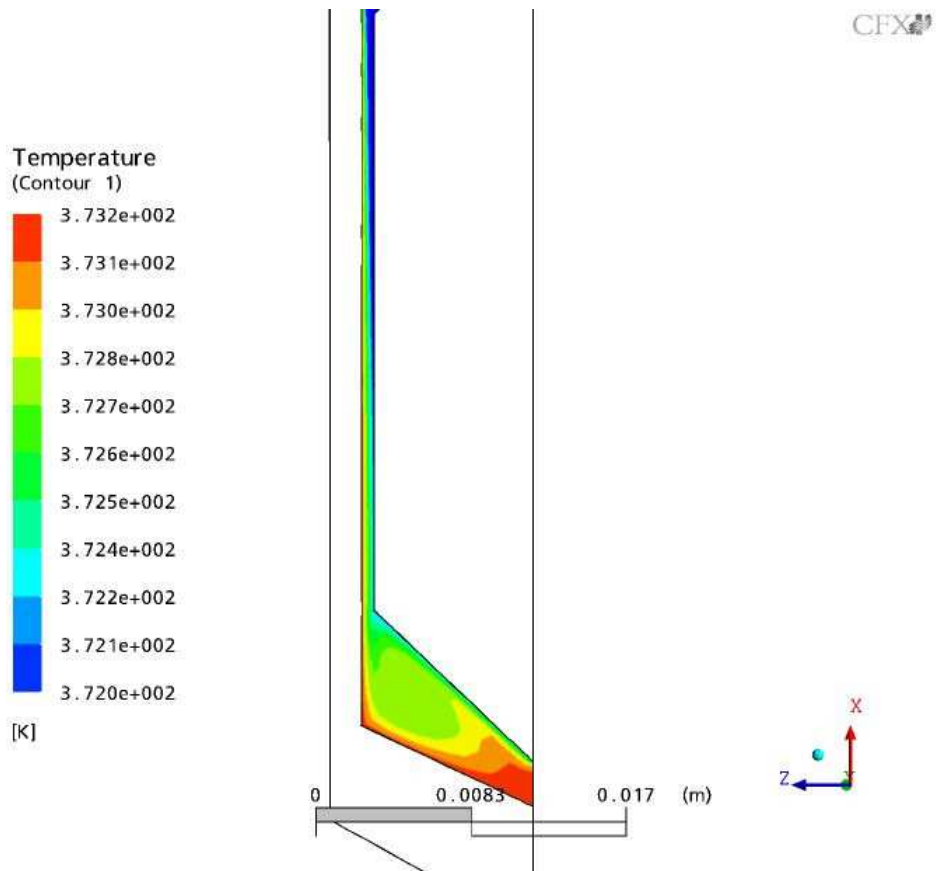


Figura 9.4: Perfil detalhado de temperaturas do fluido a 150 rpm

Os perfis de temperatura mostram que na maior rotação (150 rpm) as temperaturas em torno do cilindro são menores, e conseqüentemente a viscosidade é maior.

Os resultados desta simulação apresentam a mesma tendência observada experimentalmente, ou seja, a rotação do cilindro influencia na transferência de calor.