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ANEXO A LEI DE CONTATO DE HERTZ

A primeira tentativa de se incorporar uma teoria de indentação local foi baseada em um esquema sugerido por Hertz que percebeu o contato entre dois corpos como um problema equivalente em eletrostática (Goldsmith, 1960). O uso da lei de Hertz, além do limite da sua validade, tem sido justificado pois, esta consegue prever de forma acurada a maioria dos parâmetros de impacto que podem ser verificados experimentalmente (Goldsmith, 1960).

Derivação da Lei de Contato de Hertz (Goldsmith, 1960):

Equações das duas superfícies inicialmente no ponto 0 (como pode ser visto na figura A.1):

$$Z_1 = A_1 X_1^2 + B_1 Y_1^2 + C_1 X_1 Y_1 \quad , \quad Z_2 = A_2 X_1^2 + B_2 Y_1^2 + C_2 X_1 Y_1 \quad (\text{A.1})$$

Através da rotação dos eixos x_1 - y_1 os termos $x_1 y_1$ podem ser eliminados da equação acima.

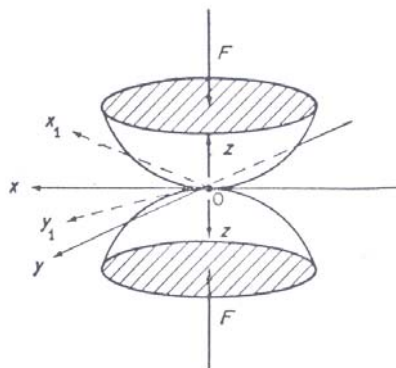


Figura A.1 – Coordenadas das duas superfícies inicialmente no ponto 0.

A distância d que separa a superfície não deformada ao longo de uma linha paralela ao eixo z é:

$$d = Z_1 + Z_2 = AX^2 + BY^2 \quad (\text{A.2})$$

As constantes A e B são relacionadas ao raio principal de curvatura R_1 e $\overline{R_1}$ para o corpo 1 e R_2 e $\overline{R_2}$ para o corpo 2 pelas seguintes equações:

$$2(B + A) = \frac{1}{R_1} + \frac{1}{\overline{R_1}} + \frac{1}{R_2} + \frac{1}{\overline{R_2}} \quad (\text{A.3})$$

$$2(B - A) = \left[\left(\frac{1}{R_1} - \frac{1}{\overline{R_1}} \right)^2 + \left(\frac{1}{R_2} - \frac{1}{\overline{R_2}} \right)^2 + 2 \left(\frac{1}{R_1} - \frac{1}{\overline{R_1}} \right) \left(\frac{1}{R_2} - \frac{1}{\overline{R_2}} \right) \cos 2\beta \right]^{1/2} \quad (\text{A.4})$$

β é o ângulo formado pelos dois planos normais contendo as duas curvaturas $1/R_1$ e $1/R_2$, respectivamente. Este ângulo pode ser eliminado pela substituição de um ângulo auxiliar θ , definido como $\cos \theta = \frac{B - A}{B + A}$ o que permite escrever a expressão anterior como:

$$2A \operatorname{cosec}^2 \frac{1}{2}\theta = 2B \sec^2 \frac{1}{2}\theta = \frac{1}{R_1} + \frac{1}{\overline{R_1}} + \frac{1}{R_2} + \frac{1}{\overline{R_2}} \quad (\text{A.5})$$

Quando uma força F é aplicada aos dois corpos ao longo do eixo z uma pequena seção da superfície de cada corpo na vizinhança do ponto de contato se deformará em uma figura de plano localizada no plano x - y . As regiões de contato e separação das superfícies são distinguidas pela seguinte relação:

$$W_1 + W_2 = \alpha - AX^2 - BY^2 \text{ e } W_1 + W_2 > \alpha - AX^2 - BY^2 \quad (\text{A.6})$$

onde, W_1 e W_2 denotam os deslocamentos de dois corpos na direção z . O termo α é o valor de $W_1 + W_2$ ao longo do eixo z e representa a compressão relativa máxima dos corpos.

Uma adaptação da teoria da elasticidade para o deslocamento devido a uma força concentrada num corpo semi-infinito para o deslocamento W no plano $z = 0$ leva a:

$$w \Big|_{z=0} = \frac{1 - \mu^2}{E\pi} \iint \frac{p dx dy}{s} = \delta \iint \frac{p dx dy}{s} \quad (\text{A.7})$$

$$\text{onde } \delta \equiv \frac{1 - \mu^2}{E\pi}$$

Substituindo a equação A.7 na equação A.6 temos:

$$\begin{aligned} W_1 + W_2 &= \alpha - AX^2 - BY^2 = \left[\frac{1 - \mu_1^2}{E_1\pi} + \frac{1 - \mu_2^2}{E_2\pi} \right] \iint \frac{p dx dy}{s} \\ &= [\delta_1 + \delta_2] \iint \frac{p dx dy}{s} \end{aligned} \quad (\text{A.8})$$

onde a pressão p desconhecida deve ser deduzida.

A tensão de compressão ou distribuição de pressão será assumida na forma de um potencial o qual deve satisfazer as seguintes condições de equilíbrio dos dois corpos (Goldsmith, 1960):

- a) A força F aplicada total deve ser igual a força total resistida pela componente vertical da pressão dentro da área de contato, ou $F = \iint p z dx dy$.
- b) No interior de cada corpo, fora da região de contato, os deslocamentos são governados pela equação com os termos da aceleração igualados a zero. As tensões decrescem rapidamente com o incremento da distância em relação à área de contato.
- c) Os componentes de deslocamento desaparecem no infinito. Na prática tais deslocamentos podem ser desprezados em grandes distâncias da área de contato quando comparadas as suas dimensões.

- d) Ao longo da superfície dos corpos: (1) as tensões de cisalhamento τ_{xz} e τ_{yz} devem desaparecer; (2) as tensões normais fora da região de contato devem desaparecer, e (3) as tensões normais agindo nos dois corpos devem ser balanceadas dentro da área de contato.

A equação 4.9 sugere que a forma da pressão potencial seja uma função quadrática de x e y . As condições a e d são satisfeitas pela escolha de um potencial que resulte de uma elipsóide de densidade ρ e formato dado por:

$$\frac{x^2}{a^2} + \frac{y^2}{b^2} + \frac{z^2}{c^2} = 1 \quad (\text{A.9})$$

no plano $z = 0$.

A distribuição da pressão na área de contato é dada por:

$$p = 2\rho c \sqrt{1 - \frac{x^2}{a^2} - \frac{y^2}{b^2}} \quad (\text{A.10})$$

A forma apropriada do potencial é então dada por:

$$\iint \frac{p dx dy}{s} = \pi(\rho c) ab \int_v^\infty \frac{1 - \frac{x^2}{a^2 + \xi} - \frac{y^2}{b^2 + \xi}}{\sqrt{[(a^2 + \xi)(b^2 + \xi)\xi]}} d\xi \quad (\text{A.11})$$

onde v é a raiz positiva da equação:

$$\frac{x^2}{a^2 + v} + \frac{y^2}{b^2 + v} + \frac{z^2}{v} = 1 \quad (\text{A.12})$$

quando o potencial é aplicado fora da região de contato, e $v = 0$ dentro do domínio.

Das equações A.10 a A.12, é possível deduzir as seguintes expressões:

$$p = \frac{3F}{2\pi ab} \sqrt{\left(1 - \frac{x^2}{a^2} - \frac{y^2}{b^2}\right)} \quad (\text{A.13})$$

$$e \quad \alpha - Ax^2 - By^2 = \frac{3}{4}[\delta_1 + \delta_2]F \int_0^\infty \frac{1 - \frac{x^2}{a^2 + \xi} - \frac{y^2}{b^2 + \xi}}{\sqrt{[(a^2 + \xi)(b^2 + \xi)\xi]}} d\xi \quad (\text{A.14})$$

Para se assegurar a aplicabilidade da equação acima, os coeficientes dos termos semelhantes devem ser iguais:

$$\alpha = \frac{3}{4}F[\delta_1 + \delta_2] \int_0^\infty \frac{d\xi}{\sqrt{[(a^2 + \xi)(b^2 + \xi)\xi]}} \quad (\text{A.15})$$

$$A = \frac{3}{4}F[\delta_1 + \delta_2] \int_0^\infty \frac{d\xi}{\sqrt{[(a^2 + \xi)^3(b^2 + \xi)\xi]}} \quad (\text{A.16})$$

$$B = \frac{3}{4}F[\delta_1 + \delta_2] \int_0^\infty \frac{d\xi}{\sqrt{[(a^2 + \xi)(b^2 + \xi)^3\xi]}} \quad (\text{A.17})$$

Substituindo $\bar{k} \equiv a/b$ e $\bar{\xi} \equiv \frac{\xi}{a^2}$ nas equações acima:

$$\alpha = \frac{3}{4}F[\delta_1 + \delta_2] \int_0^\infty \frac{d\bar{\xi}}{\sqrt{[(1 + \bar{\xi})(\bar{k}^{-2} + \bar{\xi})\bar{\xi}]}} \quad (\text{A.18})$$

$$A = \frac{3}{4} \frac{F}{a^3} [\delta_1 + \delta_2] \int_0^\infty \frac{d\bar{\xi}}{\sqrt{[(1 + \bar{\xi})^3(\bar{k}^{-2} + \bar{\xi})\bar{\xi}]}} \equiv \frac{3}{4} \frac{F}{a^3} [\delta_1 + \delta_2] [\phi_1 < \bar{k} >] \quad (\text{A.19})$$

$$B = \frac{3}{4} \frac{F}{a^3} [\delta_1 + \delta_2] \int_0^\infty \frac{d\bar{\xi}}{\sqrt{[(1 + \bar{\xi})(\bar{k}^{-2} + \bar{\xi})^3\bar{\xi}]}} \equiv \frac{3}{4} \frac{F}{a^3} [\delta_1 + \delta_2] [\phi_2 < \bar{k} >] \quad (\text{A.20})$$

Dividindo as duas equações anteriores:

$$A/B = \frac{[\phi_1 < \bar{k} >]}{[\phi_2 < \bar{k} >]} \quad (\text{A.21})$$

mostrando assim que \bar{k} , ou de outra forma, a excentricidade da elipse $e = \sqrt{1 - \bar{k}^{-2}}$ é função apenas da razão A/B. ϕ_1 e ϕ_2 podem ser calculados por polinomiais de Lagrange. As dimensões de contato da elipse podem então ser expressas por:

$$a = q_a \sqrt[3]{\frac{3F(\delta_1 + \delta_2)}{4(A+B)}} \quad \text{e} \quad b = q_b \sqrt[3]{\frac{3F(\delta_1 + \delta_2)}{4(A+B)}} \quad (\text{A.22})$$

onde $q_a = (\phi_1 + \phi_2)^{1/3}$ e $q_b = \frac{q_a}{k}$

Assim, uma combinação das equações A.18 e A.22 resulta na lei de contato de Hertz:

$$F = k_2 \alpha^{2/3} \quad (\text{A.23})$$

$$\text{onde } k_2 = \frac{4}{3} \frac{q_a^{3/2}}{(\delta_1 + \delta_2) \sqrt{(A+B)}} \equiv \frac{4}{3} \frac{q_k}{(\delta_1 + \delta_2) \sqrt{(A+B)}} \quad (\text{A.24})$$

$$\left[\int_0^\infty \frac{d\bar{\xi}}{\sqrt{[(1 + \bar{\xi})(\bar{k}^{-2} + \bar{\xi})\bar{\xi}]}} \right]^{3/2}$$

A pressão máxima ocorre no centro da elipse e pode ser calculada a partir da seguinte equação:

$$p_m = \frac{3F}{2\pi ab} \quad (\text{A.25})$$

A lei de contato de Hertz pode ser usada diretamente para se determinar o impacto em corpos elásticos quando as vibrações produzidas por colisões podem ser desprezadas. Através da segunda Lei de Newton, a força de impacto que age nos corpos em impacto é dada por:

$$F = -m_1 \ddot{w}_2 = -m_2 \ddot{w}_2 = -\frac{m_1 m_2}{m_1 + m_2} \ddot{\alpha} \quad (\text{A.26})$$

onde $\ddot{\alpha} = \ddot{w}_1 + \ddot{w}_2 \Big|_{\substack{x=0 \\ y=0}}$

Substituindo a equação de contato de Hertz na equação anterior e integrando com as seguintes condições iniciais $\dot{\alpha} = v_0, \alpha = 0$:

$$\frac{m_1 m_2}{m_1 + m_2} \ddot{\alpha} = -k_2 \alpha^{3/2} \quad \text{ou} \quad \ddot{\alpha} = -k_1 k_2 \alpha^{3/2} \quad \text{com} \quad k_1 = \frac{m_1 + m_2}{m_1 m_2} \quad (\text{A.27})$$

$$\text{e} \quad \frac{1}{2} (\dot{\alpha}^2 - v_0^2) = -\frac{2}{5} k_1 k_2 \alpha^{5/2} \quad (\text{A.28})$$

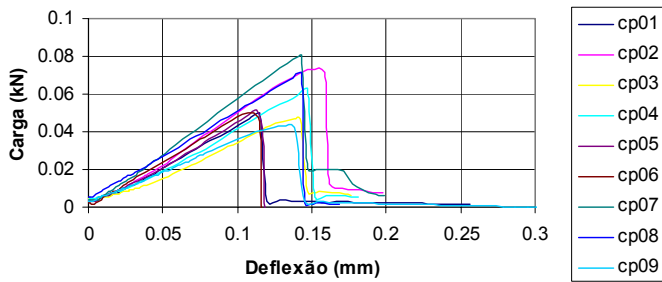
onde v_0 é a velocidade inicial. A compressão máxima α_m ocorre no instante de velocidade inicial zero $\dot{\alpha} = 0$, e conseqüentemente:

$$\alpha_m = \left[\frac{5v_0^2}{4k_1 k_2} \right]^{2/5} \quad (\text{A.29})$$

ANEXO B

CURVAS CARGA X DEFLEXÃO E TENSÃO DE FLEXÃO X DEFLEXÃO

Carga x Deflexão



Tensão de flexão x Deflexão

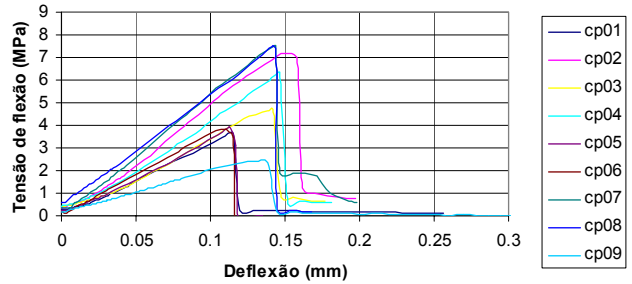
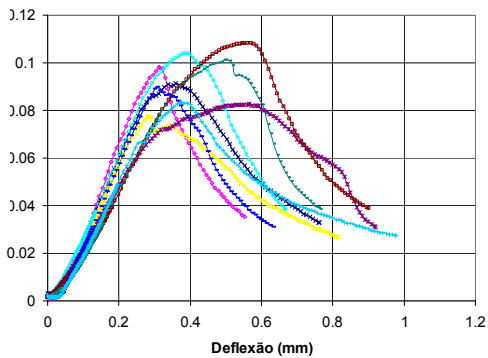


Figura B.1 – carga x deflexão e tensão de flexão x deflexão do CP0.

Carga x Deflexão



Tensão de Flexão x Deflexão

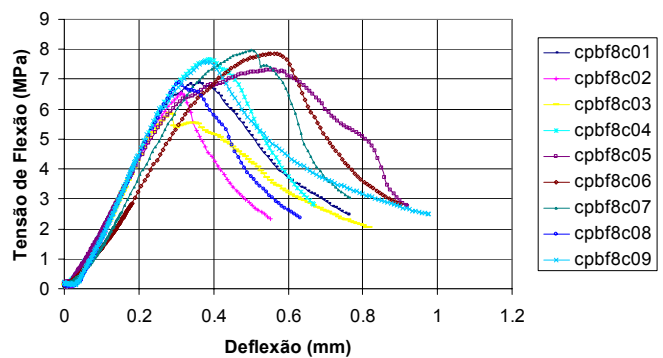


Figura B.2 – Carga x deflexão e tensão de flexão x deflexão do CPB8C.

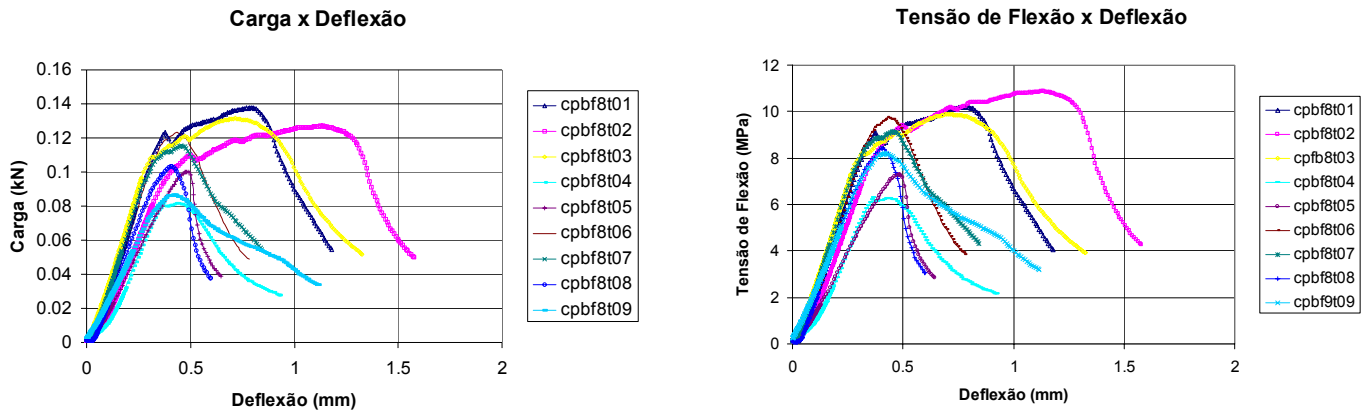
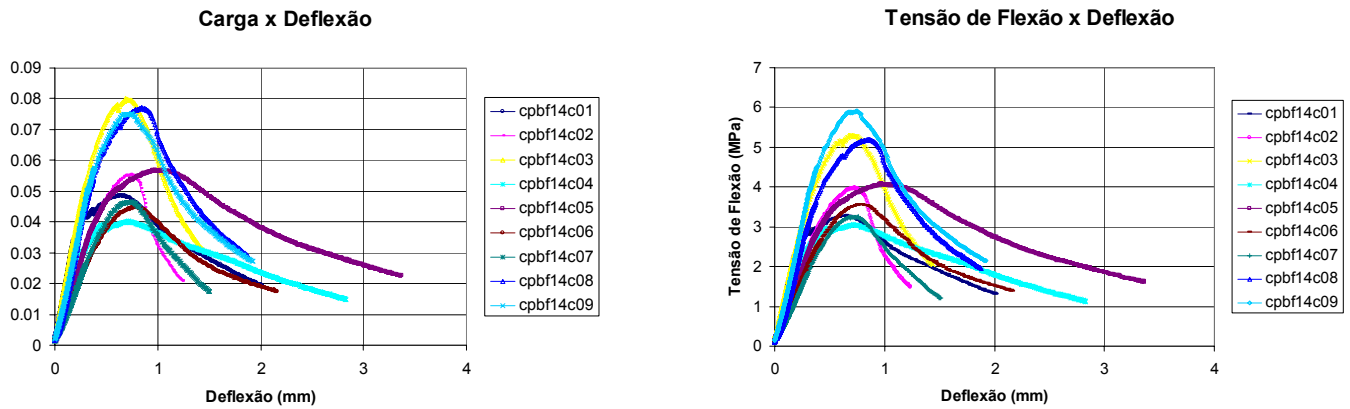


Figura B.3 - Carga x deflexão e tensão de flexão x deflexão do CPB8T



ura B.4 – Carga x deflexão e tensão de flexão x deflexão do CPB14C.

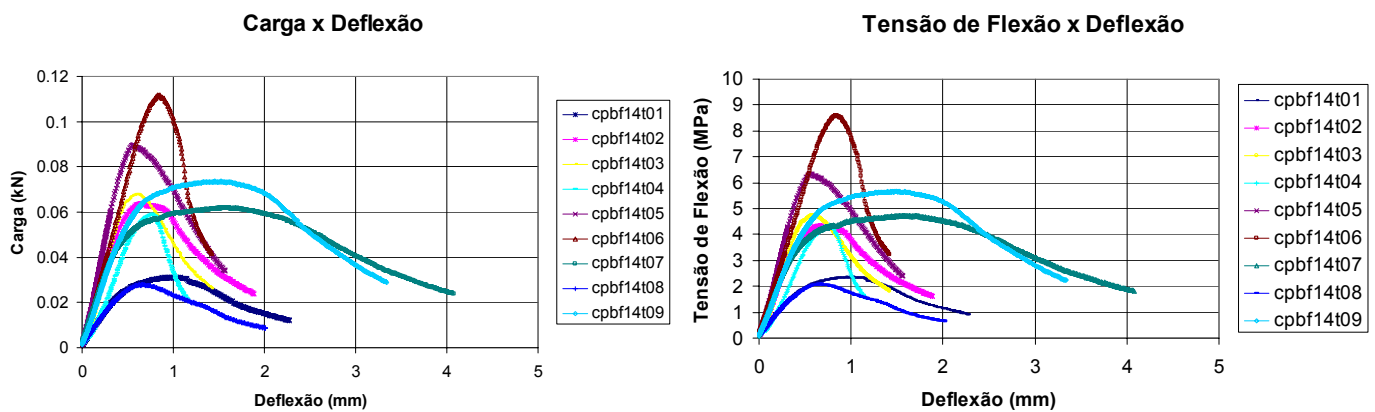


Figura B.5 – Carga x deflexão e tensão de flexão x deflexão do CPB14T.

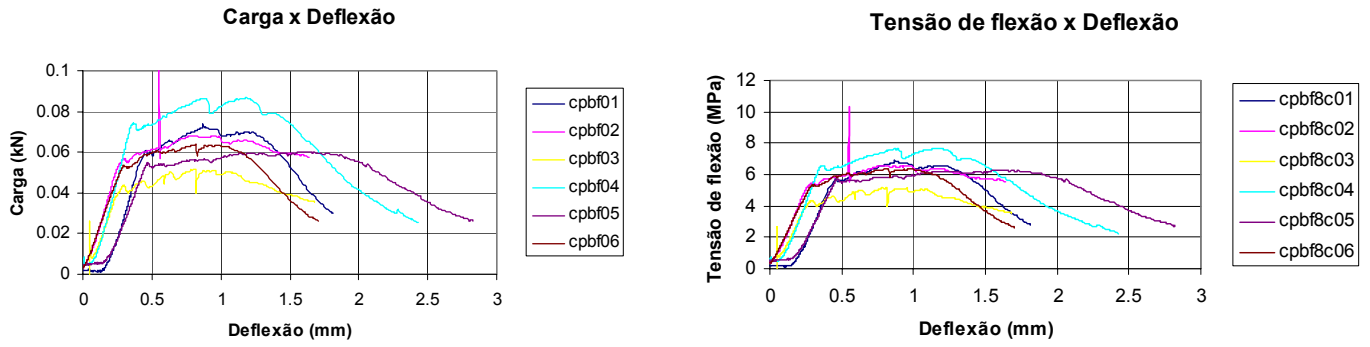


Figura B.6 – Carga x deflexão e tensão de flexão x deflexão do CPB8C (1ª série).

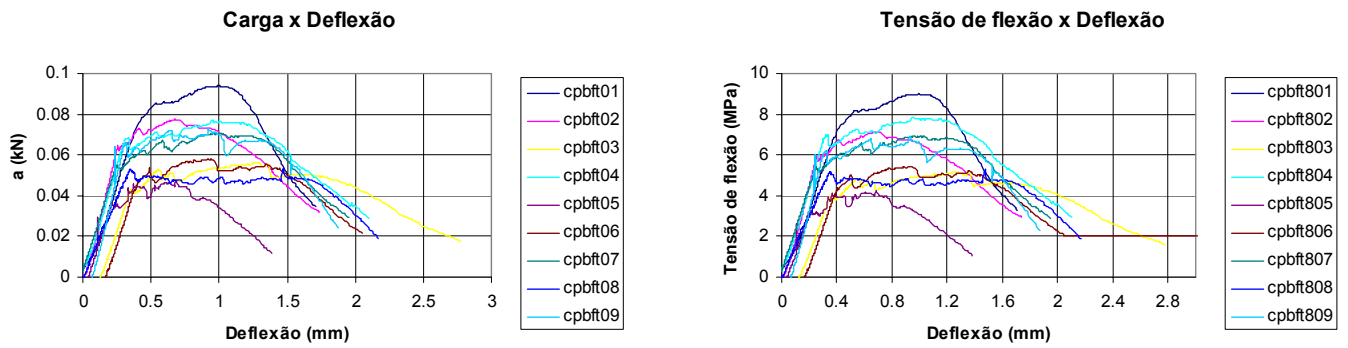


Figura B.7 – Carga x deflexão e tensão de flexão x deflexão do CPB8T (1ª série).

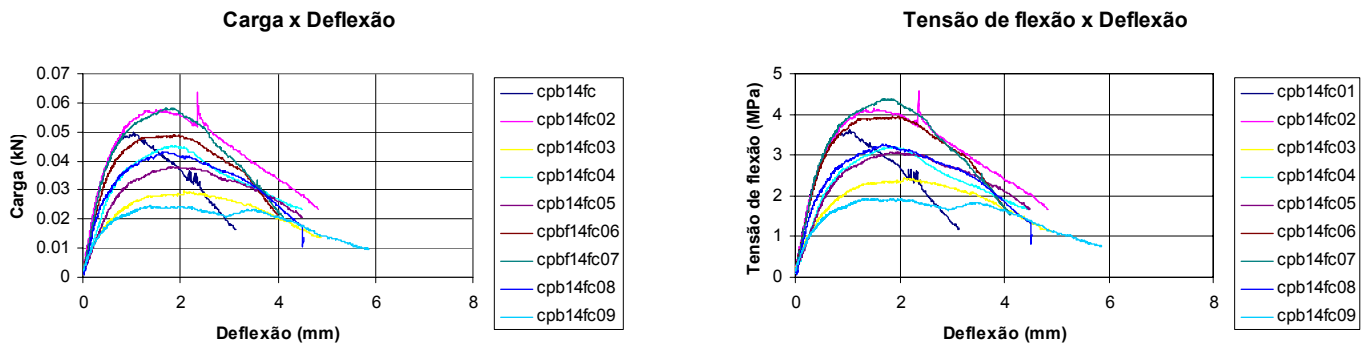


Figura B.8 – Carga x deflexão e tensão de flexão x deflexão do CPB14C (1ª série).

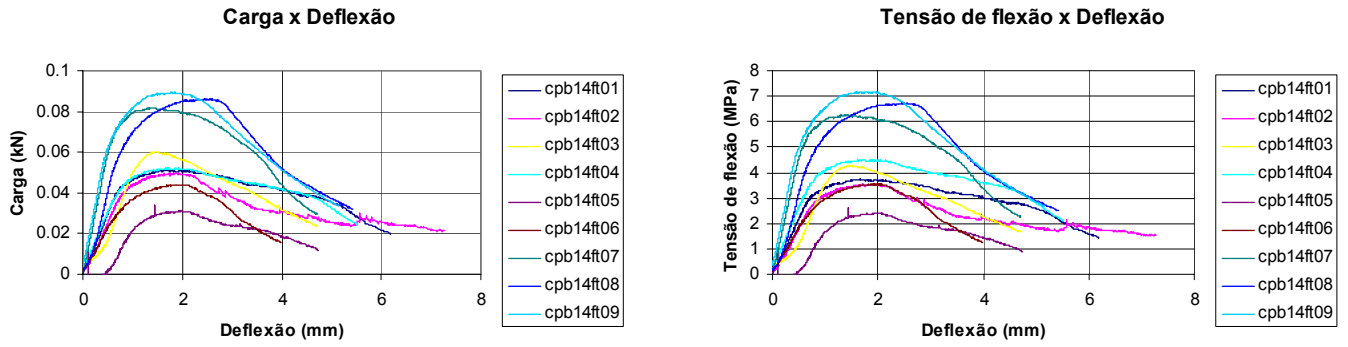
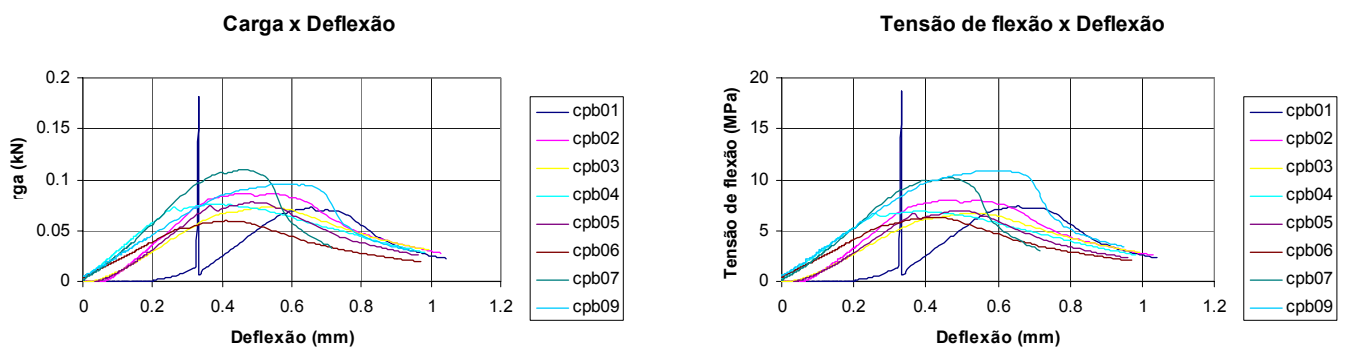


Figura B.9 – Carga x deflexão e tensão de flexão x deflexão do CPB14T (1ª série).



ura B.10 – Carga x deflexão e tensão de flexão x deflexão do CPB8C (2ª série)

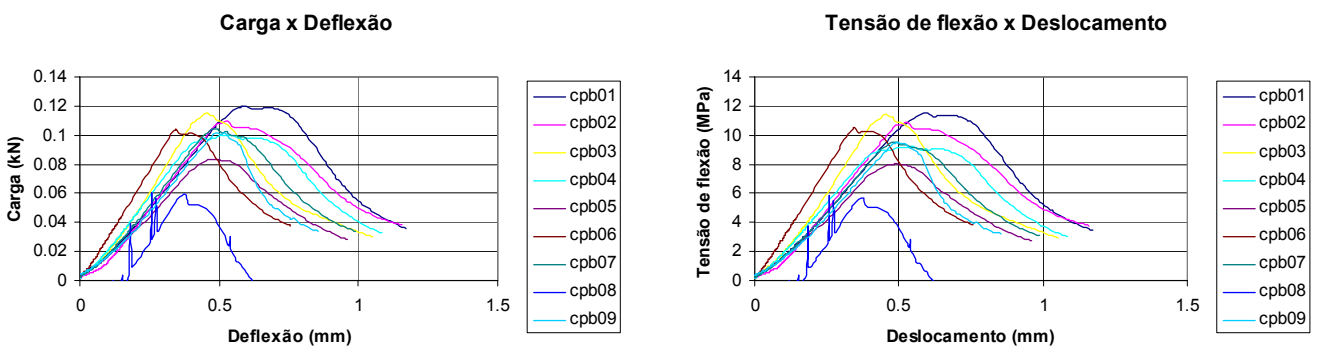


Figura B.11 – Carga x deflexão e tensão de flexão x deflexão do CPB8T (2ª série)

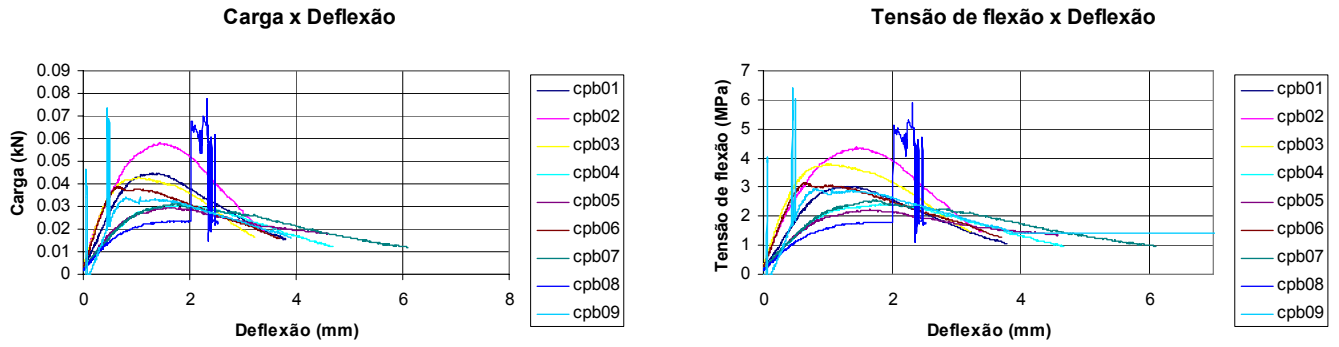


Figura B.12 – Carga x deflexão e tensão de flexão x deflexão do CPB14C (2ª serie)

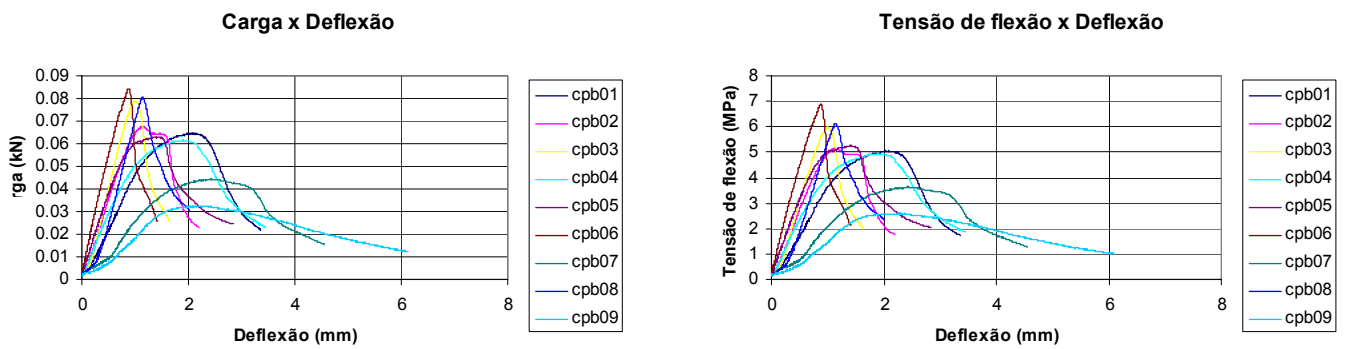


Figura B.13 – Carga x deflexão e tensão de flexão x deflexão do CPB14T (2ª serie)

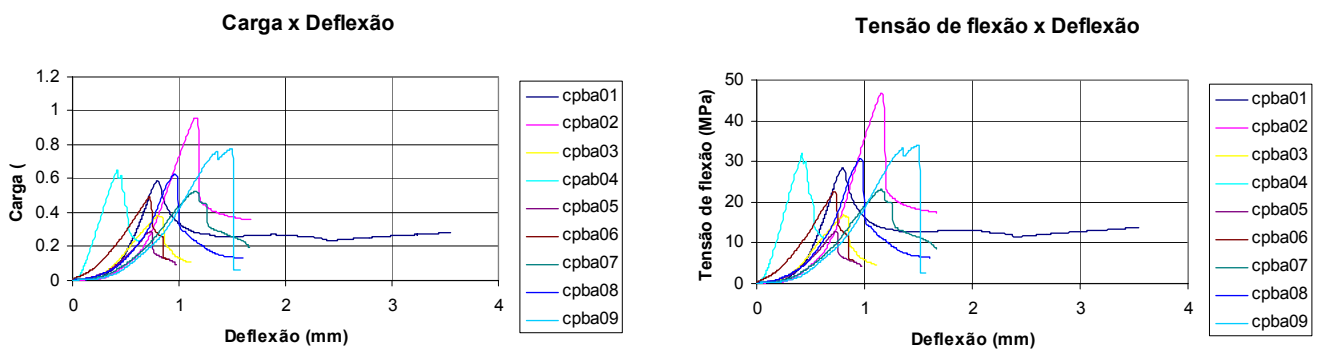


Figura B.14 – Carga x deflexão e tensão de flexão x deflexão do CPBA8.

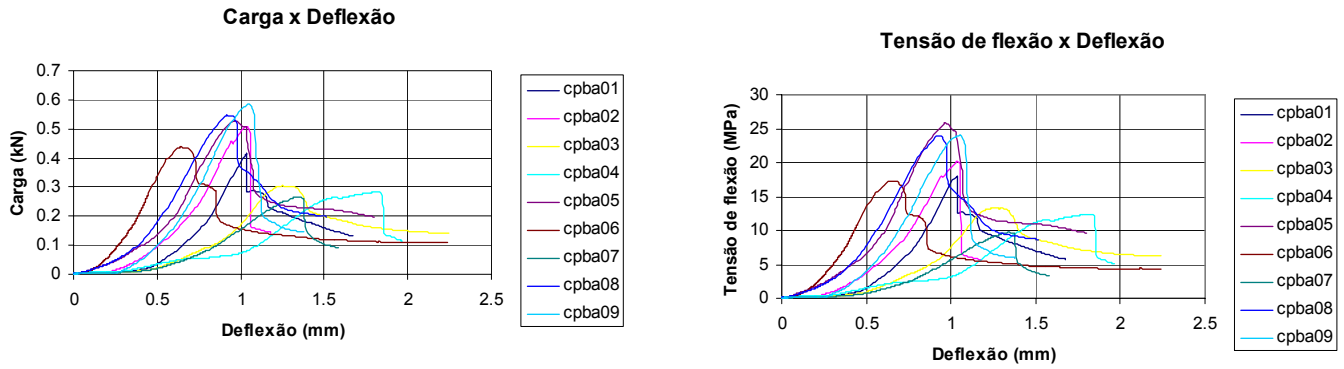
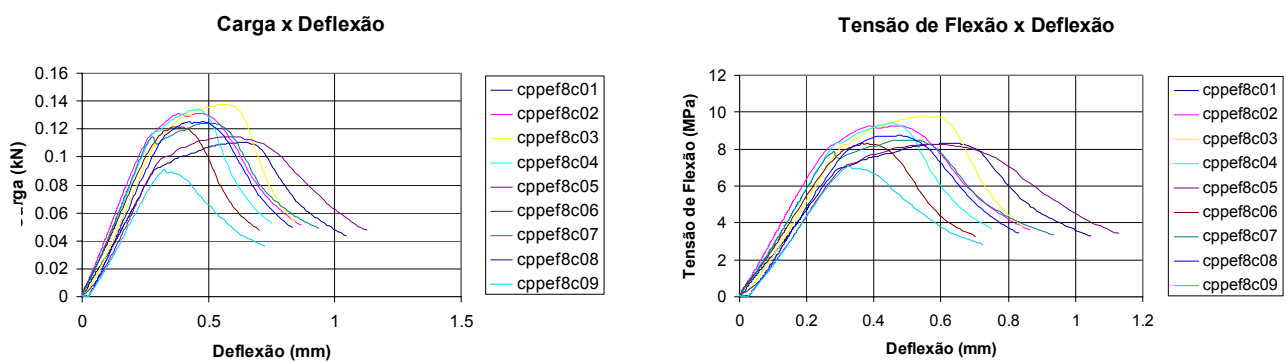


Figura B.15 – Carga x deflexão e tensão de flexão x deflexão do CPBA14.



ura B.16 – Carga x deflexão e tensão de flexão do CPE8C.

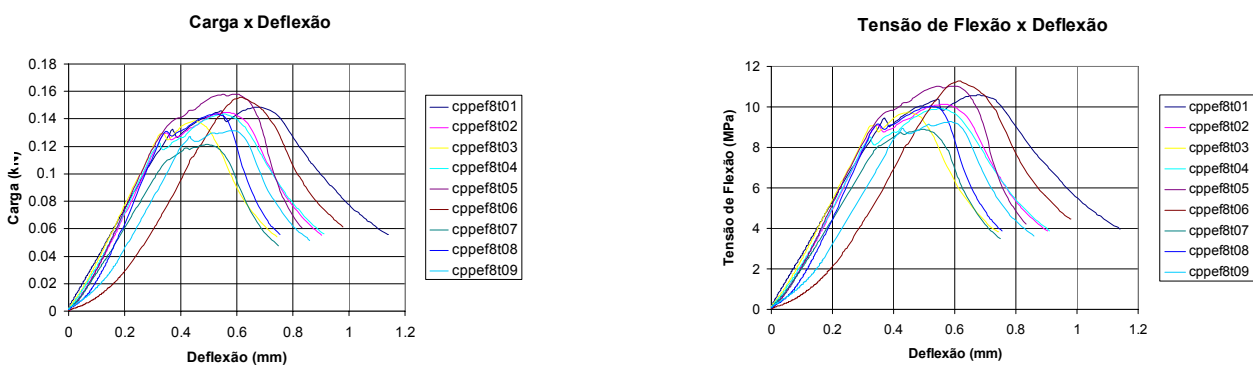


Figura B.17 – Carga x deflexão e tensão de flexão do CPE8T.

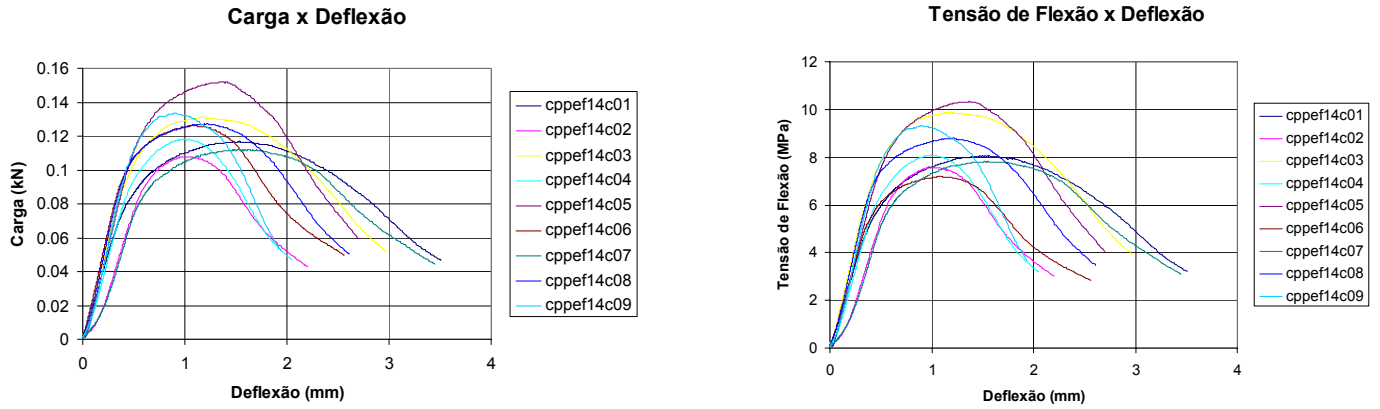


Figura B.18 – Carga x deflexão e tensão de flexão do CPE14C.

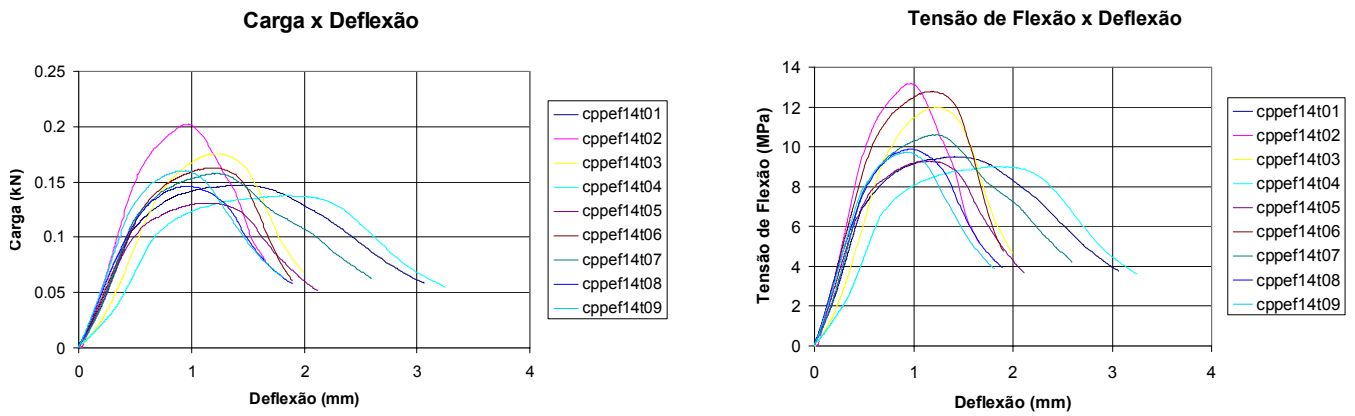


Figura B.19 – Carga x deflexão e tensão de flexão do CPE14T.

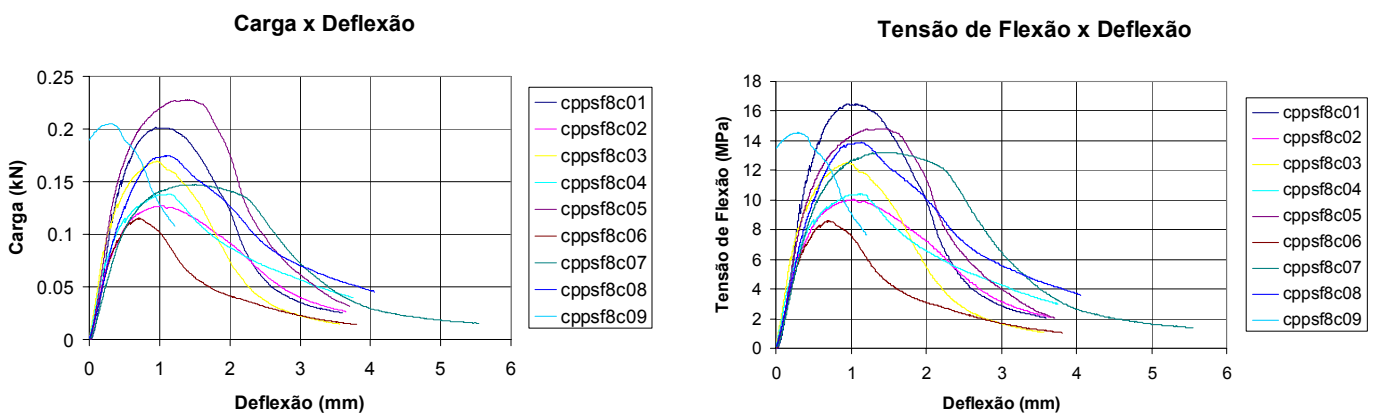


Figura B.20 – Carga x deflexão e tensão de flexão do CPS8C.

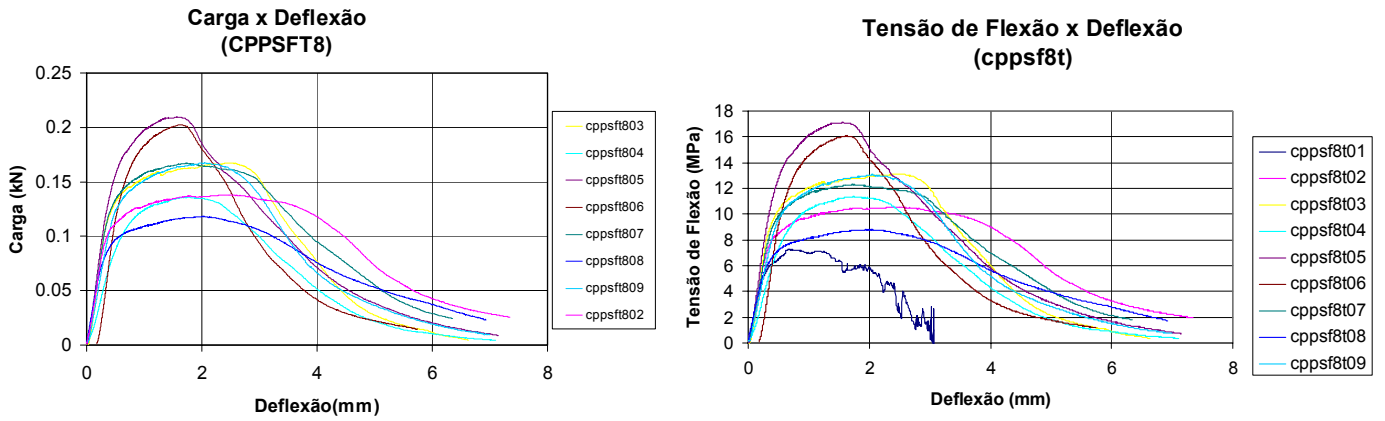


Figura B.21 – Carga x deflexão e tensão de flexão do CPS8T.

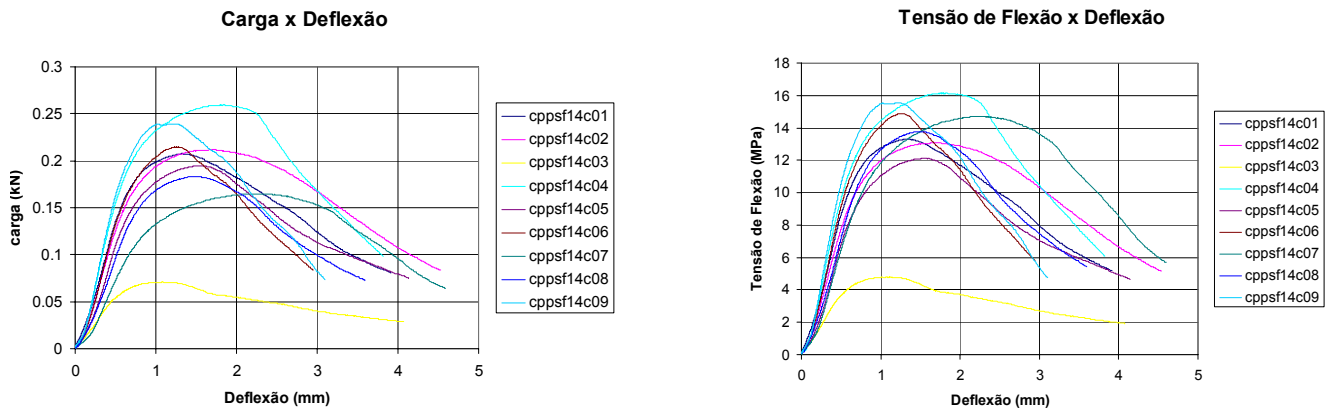


Figura B.22 – Carga x deflexão e tensão de flexão do CPS14C.

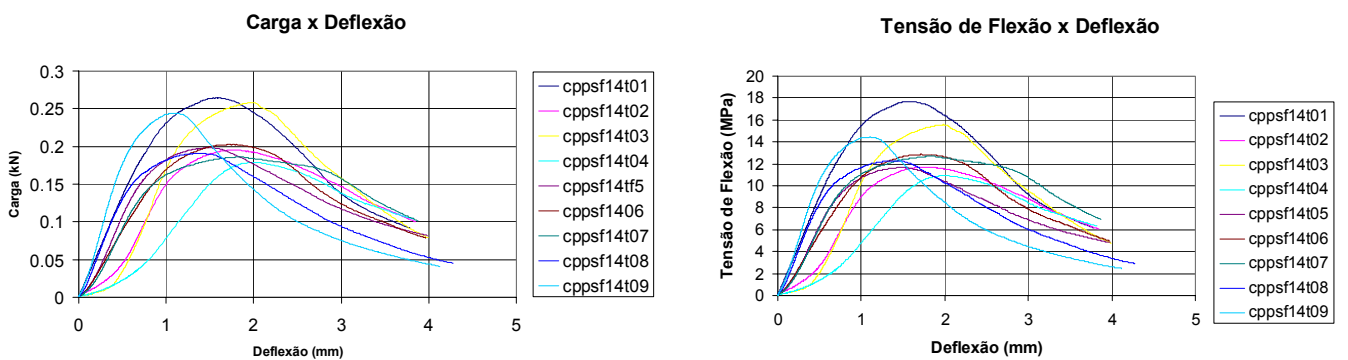


Figura B.23 – Carga x deflexão e tensão de flexão do CPS14T.

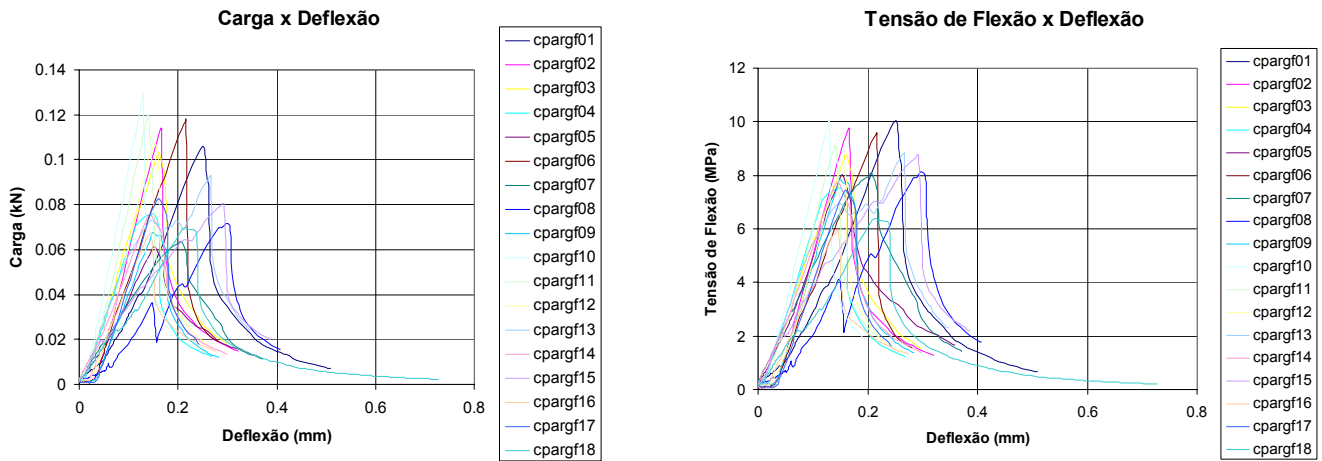
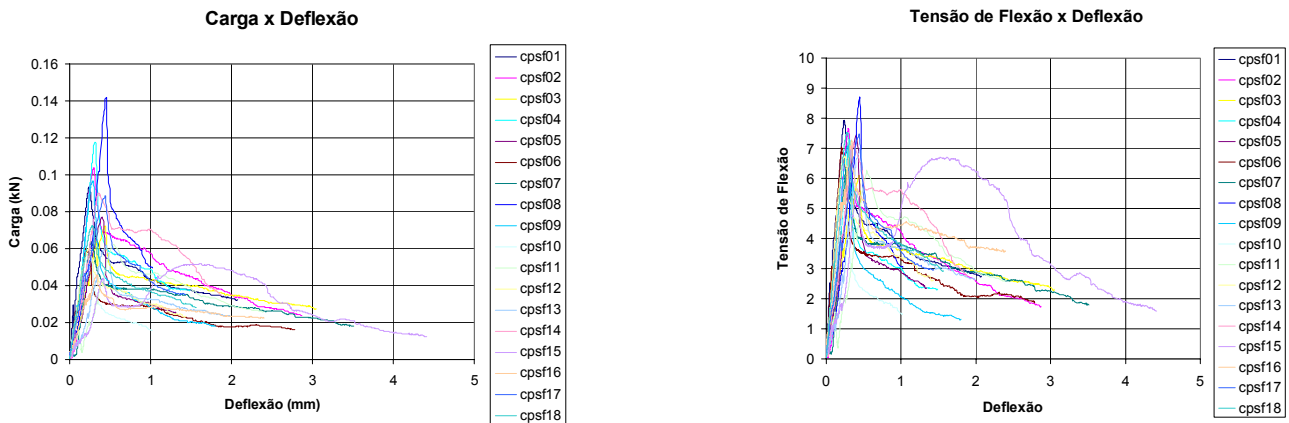


Figura B.24 – Carga x deflexão e tensão de flexão do CPARG.



ura B.25 – Carga x deflexão e tensão de flexão do CPFS.

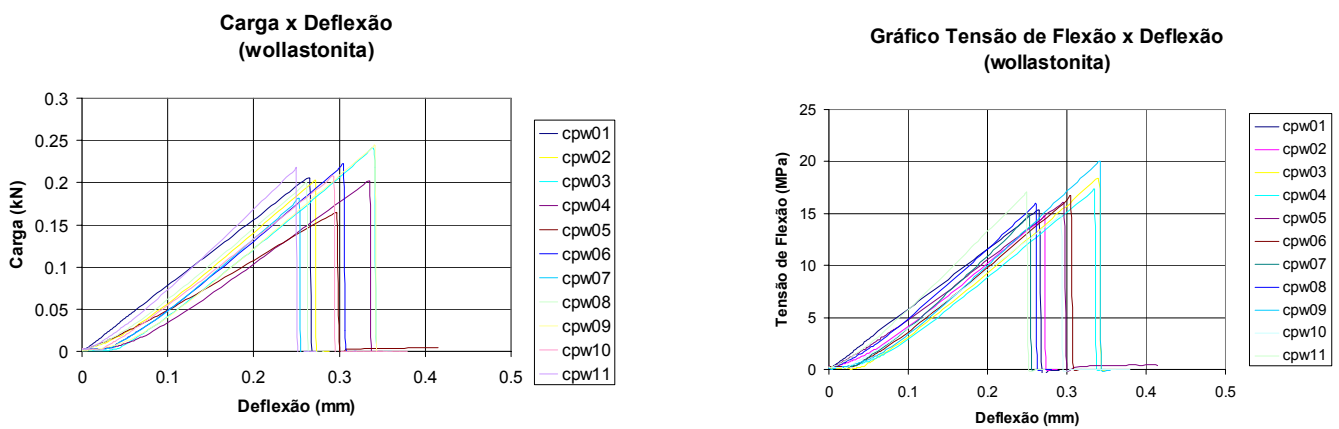


Figura B.26 – Carga x deflexão e tensão de flexão do CPW.

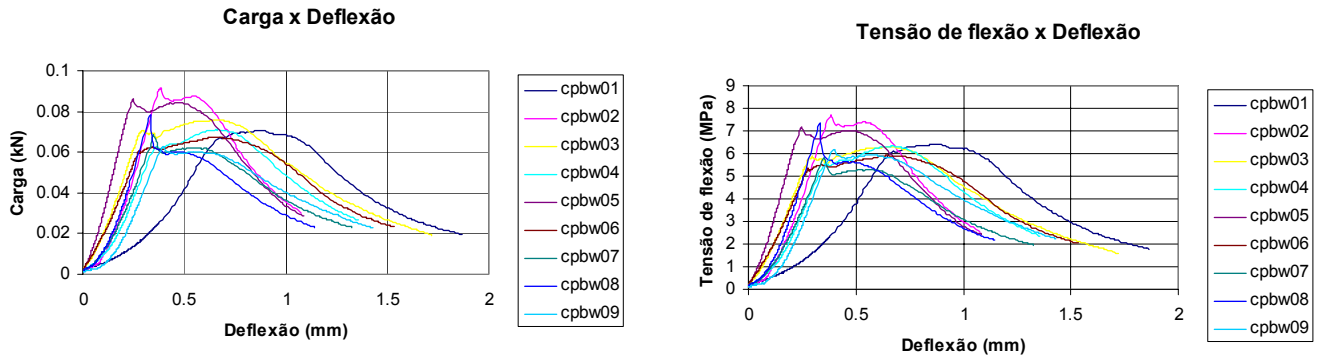


Figura B.27 – Carga x deflexão e tensão de flexão do CPBW8C.

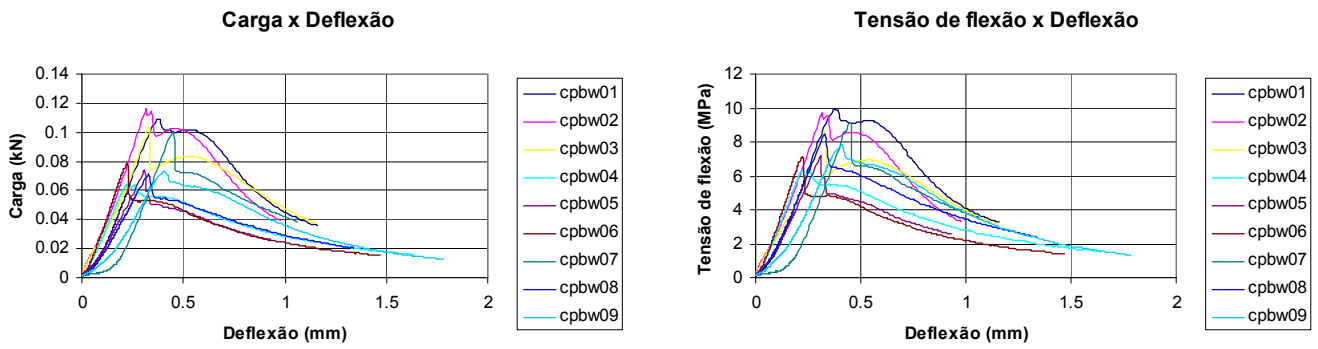


Figura B.28 – Carga x deflexão e tensão de flexão do CPBW8T.

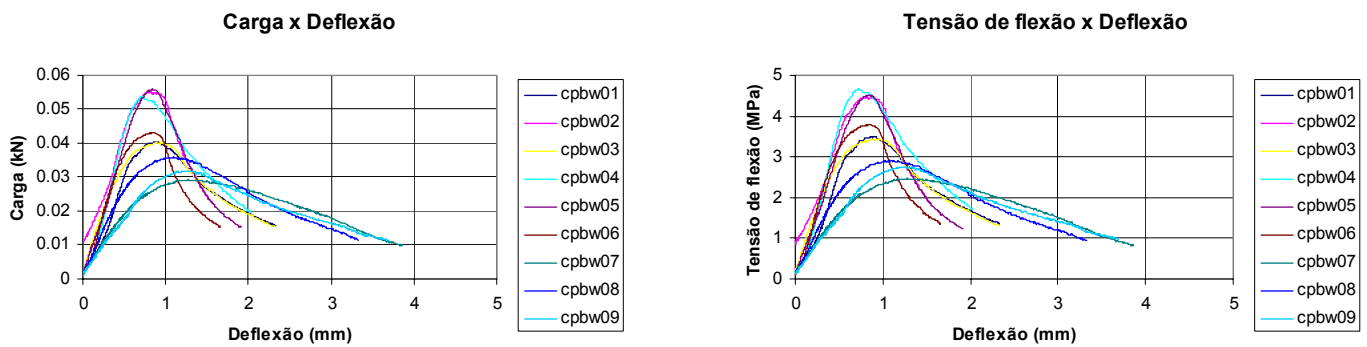


Figura B.29 – Carga x deflexão e tensão de flexão do CPBW14C.

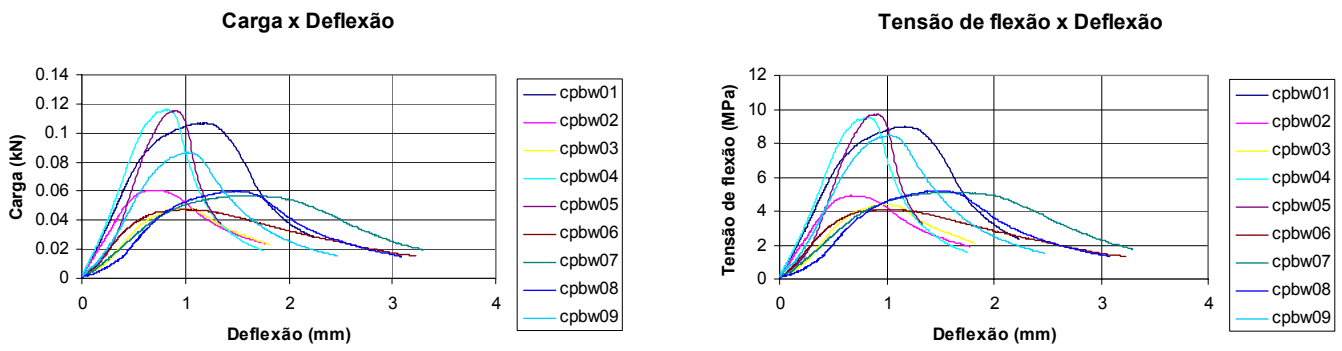
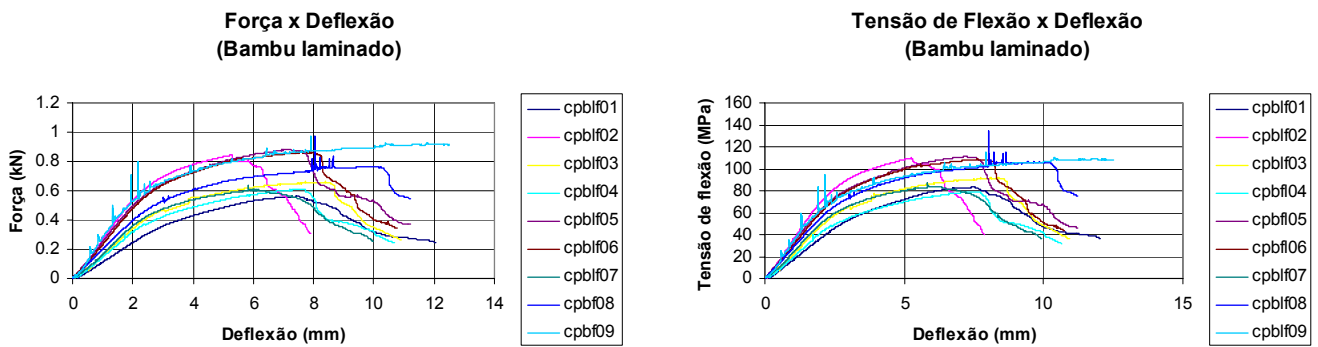


Figura B.30 – Carga x deflexão e tensão de flexão do CPBW14T.



ura B.31 – Carga x deflexão e tensão de flexão do BL.

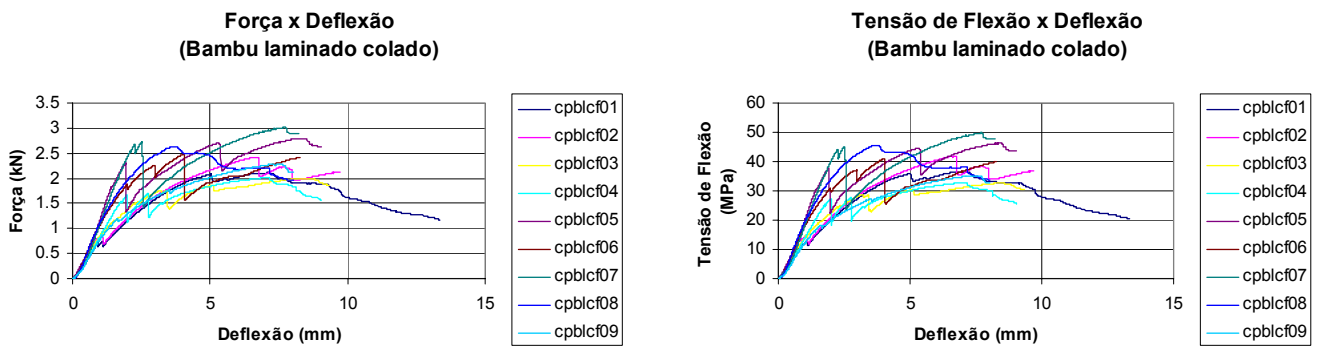


Figura B.32 – Carga x deflexão e tensão de flexão do BLC.

ANEXO C DETERMINAÇÃO DA TENSÃO DE ADESÃO INTERFACIAL E COMPRIMENTO CRÍTICO

CPBS8T:

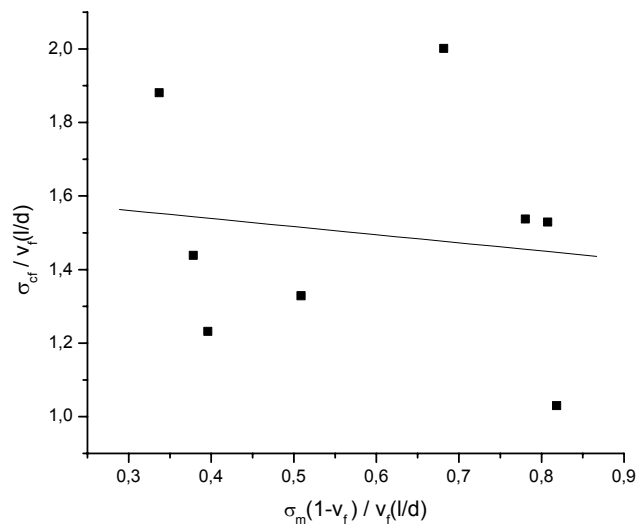


Figura C.1 – Determinação da equação do módulo de ruptura para o CPS8C.

Dessa maneira a equação 6.2 toma a seguinte forma:

$$\sigma_{cf} = -0,22006\sigma_m(1 - V_f) + 1,62689V_f(l/d) \quad (C.1)$$

Possibilitando assim o cálculo de τ e L_c :

$$0,82\tau = 1,62689 \Rightarrow \tau = 1,98MPa \text{ e através da equação 6.1, } L_c = 1,8 \text{ mm}$$

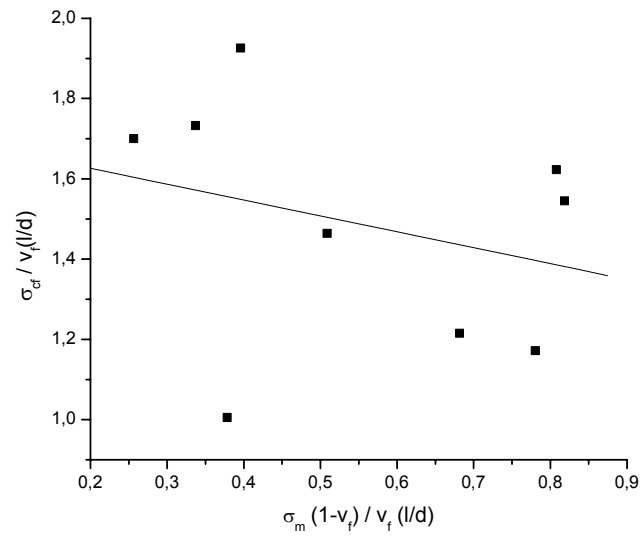
CPS8C:

Figura C.2 – Determinação da equação do módulo de ruptura para o CPS8C.

Equação da reta:

$$\sigma_{cf} = -0,39588\sigma_m(1 - V_f) + 1,70536V_f(l/d) \quad (C.2)$$

Cálculo de τ e L_c :

$$0,82\tau = 1,70536 \Rightarrow \tau = 2,08\text{MPa} \text{ e através da equação 6.1, } L_c = 1,72 \text{ mm}$$

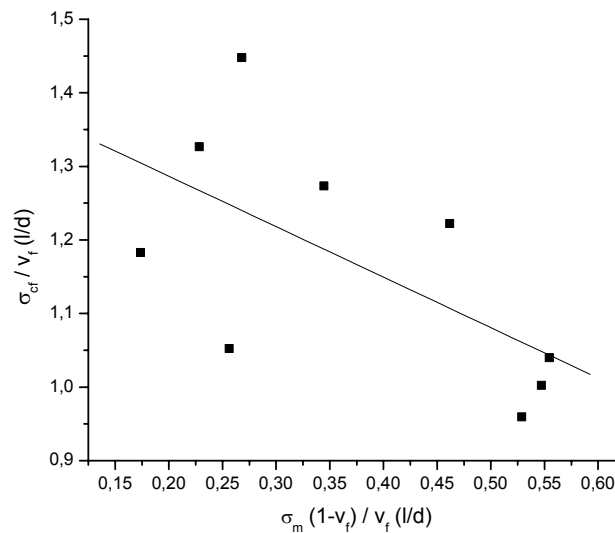
CPS14T:

Figura C.3 – Determinação da equação do módulo de ruptura para o CPS14T.

Equação da reta:

$$\sigma_{cf} = -0,68582\sigma_m(1 - V_f) + 1,42376V_f(l/d) \quad (C.3)$$

Cálculo de τ e L_c :

$$0,82\tau = 1,42376 \Rightarrow \tau = 1,73MPa \text{ e através da equação 6.1, } L_c = 2,06 \text{ mm}$$

CPS14C:

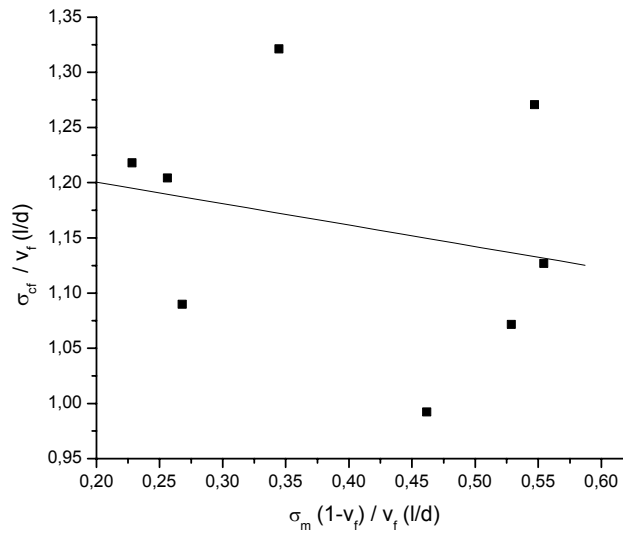


Figura C.4 – Determinação da equação do módulo de ruptura para o CPS14C.

Equação da reta:

$$\sigma_{cf} = -0,1943\sigma_m(1 - V_f) + 1,23931V_f(l/d) \quad (C.4)$$

Cálculo de τ e L_c :

$$0,82\tau = 1,23931 \Rightarrow \tau = 1,51MPa \text{ e através da equação 6.1, } L_c = 2,36 \text{ mm}$$

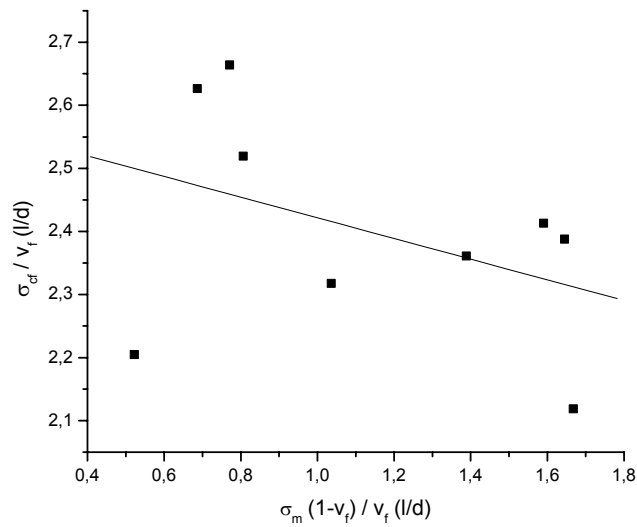
CPE8T:

Figura C.5 – Determinação da equação do módulo de ruptura para o CPE8T.

Equação da reta:

$$\sigma_{cf} = -0,16387\sigma_m(1 - V_f) + 2,5855V_f(l/d) \quad (C.5)$$

Cálculo de τ e L_c :

$$0,82\tau = 2,5855 \Rightarrow \tau = 3,15\text{MPa} \text{ e através da equação 6.1, } L_c = 1,27 \text{ mm}$$

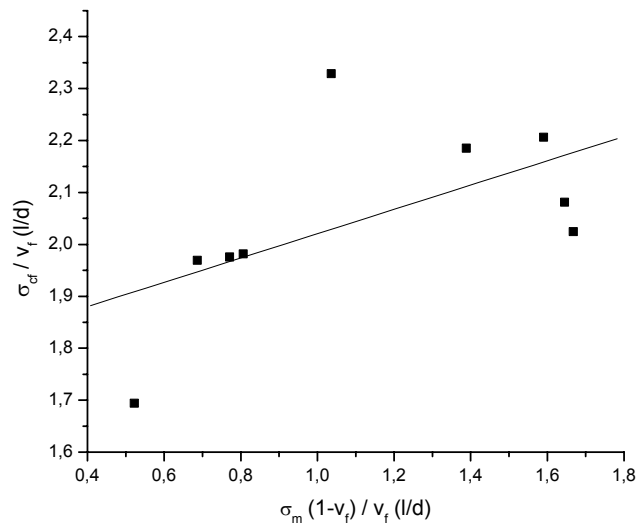
CPE8C:

Figura C.6 – Determinação da equação do módulo de ruptura para o CPE8C.

Equação da reta:

$$\sigma_{cf} = 0,23402\sigma_m(1 - V_f) + 1,78663V_f(l/d) \quad (C.6)$$

Cálculo de τ e L_c :

$0,82\tau = 1,78663 \Rightarrow \tau = 2,17MPa$ e através da equação 6.1, $L_c = 1,84$ mm

CPE14T:

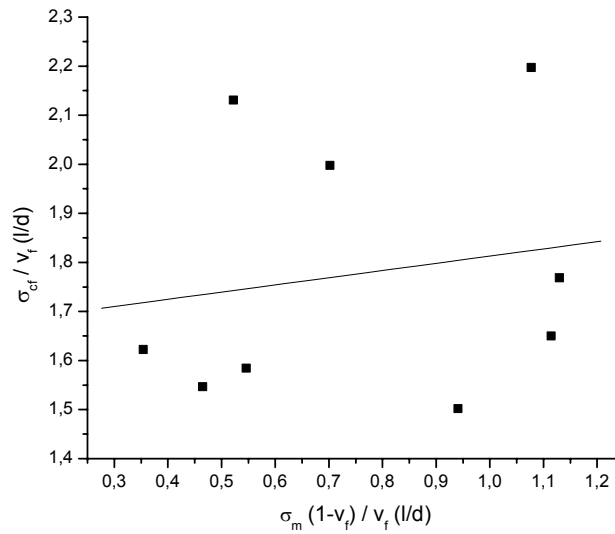


Figura C.7 – Determinação da equação do módulo de ruptura para o CPE14T.

Equação da reta:

$$\sigma_{cf} = 0,14662\sigma_m(1 - V_f) + 1,66612V_f(l/d) \quad (C.7)$$

Cálculo de τ e L_c :

$0,82\tau = 1,66612 \Rightarrow \tau = 2,03MPa$ e através da equação 6.1, $L_c = 1,97$ mm.

CPE14C:

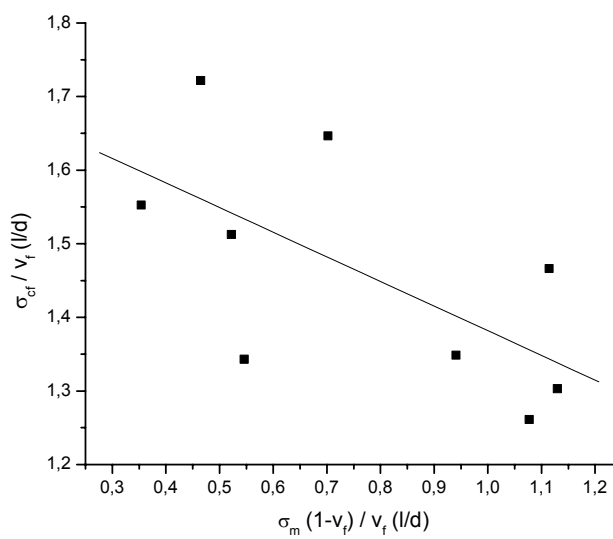


Figura C.8 – Determinação da equação do módulo de ruptura para o CPE14C.

Equação da reta:

$$\sigma_{cf} = -0,33469\sigma_m(1 - V_f) + 1,7165V_f(l/d) \quad (C.8)$$

Cálculo de τ e L_c :

$0,82\tau = 1,7165 \Rightarrow \tau = 2,09MPa$ e através da equação 6.1, $L_c = 1,91$ mm.

CPFS:

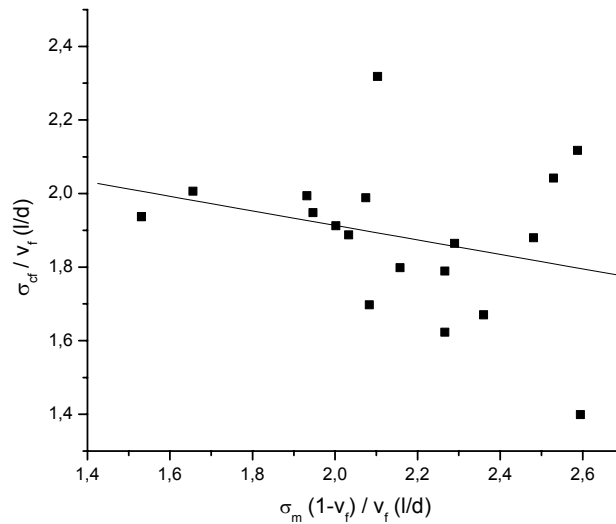


Figura C.9 – Determinação da equação do módulo de ruptura para o CPFS.

Equação da reta:

$$\sigma_{cf} = -0,19739\sigma_m(1 - V_f) + 2,30835V_f(l/d) \quad (C.9)$$

Cálculo de τ e L_c :

$0,82\tau = 2,30835 \Rightarrow \tau = 2,85MPa$ e através da equação 6.1, $L_c = 28$ mm.

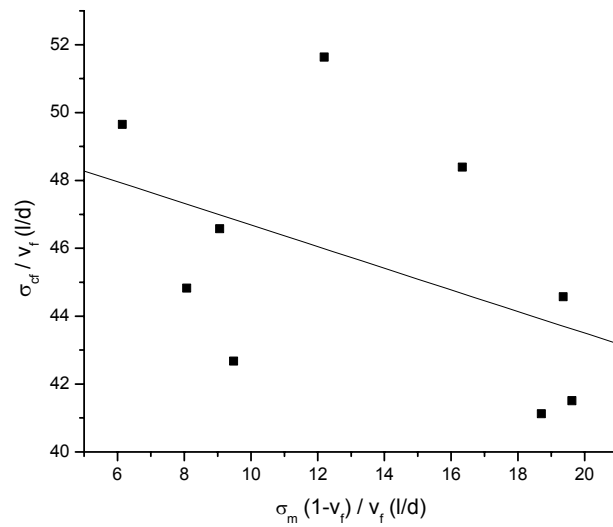
CPW:

Figura C.10 – Determinação da equação do módulo de ruptura para o CPW.

Equação da reta:

$$\sigma_{cf} = -0,31861\sigma_m(1 - V_f) + 49,87328V_f(l/d) \quad (C.10)$$

Cálculo de τ e L_c :

$0,82\tau = 49,87 \Rightarrow \tau = 60,81\text{MPa}$ e através da equação 6.1, $L_c = 32,88 \mu\text{m}$

utilizando $l = 50$ e $d = 10 \mu\text{m}$ (valores médios) e resistência à tração de 400 MPa